

Lecture – 18

*Time Response of Linear Dynamical Systems
in State Space Form*

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Solution of Linear Differential Equations

Linear systems:

Systems that obey the “Principle of superposition”.

Uniqueness Theorem:

There is only one solution for linear systems.

Solution of Homogeneous Linear Differential Equation: Scalar case

System dynamics: $\dot{x} = ax, \quad x(t_0) = x_0$

Solution: $(dx/x) = a dt$

$$\ln x = at + \ln c$$

$$\ln(x/c) = at$$

$$x = e^{at} c$$

Initial condition: $x_0 = e^{at_0} c, \quad c = e^{-at_0} x_0$

Hence,

$$x(t) = e^{a(t-t_0)} x_0$$

Solution of Homogeneous Linear Differential Equation: Scalar case

Note:

$$e^{at} = 1 + at + \frac{a^2 t^2}{2!} + \frac{a^3 t^3}{3!} + \dots$$

If $t_0 = 0$, then the solution is

$$x(t) = e^{at} x_0$$

Solution of Homogeneous Linear Differential Equations

System dynamics: $\dot{X} = AX, \quad X(t_0) = X_0$

Guess solution: $X(t) = e^{At}C, \quad C = [c_1 \quad \cdots \quad c_n]^T$

$$e^{At} \triangleq I + At + A^2t^2 / 2! + A^3t^3 / 3! + \cdots$$

Verify (substitute the guess into the differential equation)

$$\left(\frac{d}{dt} e^{At} \right) C = A(e^{At}C)$$

Solution of Homogeneous Linear Differential Equations

A Result: $e^{At} = I + At + A^2 t^2 / 2! + A^3 t^3 / 3! + \dots$

$$\frac{d}{dt} (e^{At}) = 0 + A + A^2 (2t / 2!) + A^3 (3t^2 / 3!) + \dots$$

$$= A (I + At + A^2 t^2 / 2! + A^3 t^3 / 3! + \dots)$$

$$= A e^{At}$$

i.e. $(A e^{At}) C = A (e^{At} C)$

Therefore $X(t) = e^{At} C$ is 'a' solution.

Hence, $X(t) = e^{At} C$ is 'the' solution.

Solution of Homogeneous Linear Differential Equations

- Applying the initial condition $X_0 = e^{At_0} C$
 $C = [e^{At_0}]^{-1} X_0$
- Another result: $e^{A(t_1+t_2)} = e^{At_1} e^{At_2}$
(easy to show from definition)

Taking $t_1 = t_0$ and $t_2 = -t_0$, $I = e^{At_0} e^{-At_0}$

Thus $[e^{At_0}]^{-1} = e^{-At_0}$

Finally $X(t) = e^{At} e^{-At_0} X_0 = e^{A(t-t_0)} X_0$

Solution of Non-homogeneous Linear Differential Equations

Non-homogeneous system: $\dot{X} = AX + BU$, $X(t_0) = X_0$

Solution contains two parts:

- Homogeneous solution
- Particular solution

Homogeneous solution: $X_h(t) = e^{A(t-t_0)} X_0$

Particular solution: $X_p(t) = e^{At} C(t)$

Solution of Non-homogeneous Linear Differential Equations

$$\dot{X}_p = e^{At} \dot{C} + \cancel{Ae^{At} C} = \cancel{Ae^{At} C} + BU$$

$$\dot{C} = e^{-At} BU$$

$$C(t) = \int_{t_1}^t e^{-A\tau} BU(\tau) d\tau$$

$$X_p(t) = e^{At} C(t) = e^{At} \int_{t_1}^t e^{-A\tau} BU(\tau) d\tau$$

$$= \int_{t_1}^t e^{A(t-\tau)} BU(\tau) d\tau$$

Solution of Non-homogeneous Linear Differential Equations

Complete solution: $X(t) = X_h(t) + X_p(t)$

$$= e^{A(t-t_0)} X_0 + \int_{t_1}^t e^{A(t-\tau)} BU(\tau) d\tau$$

Initial condition: At $t = t_0$

$$X_0 = X_0 + \int_{t_1}^{t_0} e^{A(t-\tau)} BU(\tau) d\tau$$

$$\int_{t_1}^{t_0} e^{A(t-\tau)} BU(\tau) d\tau = 0$$

This suggests that $t_1 = t_0$

Solution of Non-homogeneous Linear Differential Equations

Complete solution:

$$X(t) = e^{A(t-t_0)} X_0 + \int_{t_0}^t e^{A(t-\tau)} B U(\tau) d\tau$$

The integral term in the forced system solution is a *convolution integral*.

Note: If U is in feedback form ($U = -K X$)

$$\dot{X} = (A - BK) X = A_{CL} X$$

$$X(t) = e^{A_{CL}(t-t_0)} X_0$$

Solution of Non-homogeneous Linear Differential Equations: Some Comments

The solution results do not demand that $t \geq t_0$.
They are equally valid even if $t \leq t_0$.

The integral term in the forced system solution is a "convolution integral".
i.e. The contribution of input $U(t)$ is the convolution of $U(t)$ with $e^{At} B$.
Hence, the function $e^{At} B$ has the role of "impulse response" of the system
whose output is $X(t)$ and input is $U(t)$.

The solution for output $Y(t)$ is also readily available from $X(t)$ and $U(t)$:

$$Y(t) = CX(t) + DU(t)$$

Example: Motion of a car without friction

The equation of motion is

$$m \ddot{x} = f(t)$$

$$\ddot{x} = (1/m) f(t) \quad \text{Assumption: } m \text{ is constant.}$$

$$v \triangleq \dot{x}$$

$$\dot{X} = \begin{bmatrix} \dot{x} \\ \dot{v} \end{bmatrix} = \underbrace{\begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix}}_A \begin{bmatrix} x \\ v \end{bmatrix} + \underbrace{\begin{bmatrix} 0 \\ 1/m \end{bmatrix}}_B f(t), \quad X(0) = \begin{bmatrix} x_0 \\ v_0 \end{bmatrix}$$

Example: Motion of a car without friction

$$X(t) = e^{At} X_0 + \int_0^t e^{A(t-\tau)} B f(\tau) d\tau$$

$$e^{At} = I + At + A^2 \frac{t^2}{2!} + \dots = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} t = \begin{bmatrix} 1 & t \\ 0 & 1 \end{bmatrix}$$

$$e^{A(t-\tau)} = \begin{bmatrix} 1 & t-\tau \\ 0 & 1 \end{bmatrix}$$

$$e^{A(t-\tau)} B = \begin{bmatrix} 1 & t-\tau \\ 0 & 1 \end{bmatrix} \begin{bmatrix} 0 \\ 1/m \end{bmatrix} = \frac{1}{m} \begin{bmatrix} t-\tau \\ 1 \end{bmatrix}$$

Example: Motion of a car without friction

$$\begin{bmatrix} x(t) \\ v(t) \end{bmatrix} = \begin{bmatrix} x_0 + v_0 t + \frac{1}{m} \int_0^t (t - \tau) f(\tau) d\tau \\ v_0 + \frac{1}{m} \int_0^t f(\tau) d\tau \end{bmatrix} = \begin{bmatrix} x_0 + v(t)t - \frac{1}{m} \int_0^t \tau f(\tau) d\tau \\ v_0 + \frac{1}{m} \int_0^t f(\tau) d\tau \end{bmatrix}$$

Special case: $f(t)/m = a$ (constant) and $\begin{bmatrix} x_0 \\ v_0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$

$$\begin{bmatrix} x(t) \\ v(t) \end{bmatrix} = \begin{bmatrix} x_0 + (v_0 + at)t - \frac{t^2}{2} a \\ v_0 + at \end{bmatrix} = \begin{bmatrix} \frac{1}{2} at^2 \\ at \end{bmatrix}$$

Evaluation of e^{At} : A Useful Result

Problem: $\dot{X} = AX, \quad X(0) = X_0$

Solution using Laplace transform:

$$sX(s) - X_0 = AX(s)$$

$$(sI - A)X(s) = X_0$$

$$X(s) = (sI - A)^{-1} X_0$$

$$X(t) = L^{-1} \left[(sI - A)^{-1} \right] X_0$$

Solution known:

$$X(t) = e^{At} X_0$$

Comparing the two solutions:

$$e^{At} = L^{-1} \left[(sI - A)^{-1} \right]$$

Evaluation of e^{At} :
 How to compute it symbolically?

$$A = \begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1n} \\ a_{21} & a_{22} & \cdots & a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \cdots & a_{nn} \end{bmatrix}, \quad (sI - A) = \begin{bmatrix} s - a_{11} & -a_{12} & \cdots & -a_{1n} \\ -a_{21} & s - a_{22} & \cdots & -a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ -a_{n1} & -a_{n2} & \cdots & s - a_{nn} \end{bmatrix}$$

$$(sI - A)^{-1} = \frac{\text{adj}(sI - A)}{|sI - A|}$$

$$|sI - A| = s^n + a_1 s^{n-1} + a_2 s^{n-2} + \cdots + a_n$$

$$\text{adj}(sI - A) = E_1 s^{n-1} + E_2 s^{n-2} + \cdots + E_n$$

Symbolic computation of e^{At}

$$(sI - A)(sI - A)^{-1} = \frac{(sI - A)(E_1 s^{n-1} + E_2 s^{n-2} + \dots + E_n)}{s^n + a_1 s^{n-1} + a_2 s^{n-2} + \dots + a_n}$$

$$\begin{aligned} I(s^n + a_1 s^{n-1} + a_2 s^{n-2} + \dots + a_n) \\ = (sI - A)(E_1 s^{n-1} + E_2 s^{n-2} + \dots + E_n) \end{aligned}$$

$$\begin{aligned} s^n I + a_1 s^{n-1} I + a_2 s^{n-2} I + \dots + a_n I \\ = s^n E_1 + s^{n-1} (E_2 - AE_1) + \dots + s(E_n - AE_{n-1}) - AE_n \end{aligned}$$

Equate the coefficients on both sides...

Symbolic computation of e^{At}

$$E_1 = I$$

$$E_2 - AE_1 = a_1 I$$

$$E_3 - AE_2 = a_2 I$$

\vdots

$$E_n - AE_{n-1} = a_{n-1} I$$

$$-AE_n = a_n I$$

This suggests a recursive algorithm!
(provided a_1, \dots, a_n are known)



$$a_i = -\left(\frac{1}{i}\right) \text{Tr}(AE_i)$$

for $i = 1, \dots, n$

Reference: T. Kailath, Linear Systems, Prentice-Hall, 1980.

Solution of Linear Time Varying Systems

Homogeneous Linear System $\dot{X} = A(t)X$

Solution: $X(t) = \varphi(t, \tau) X(\tau)$

$\varphi(t, \tau)$: State Transition Matrix (STM)

PROPERTIES OF STM

1. It satisfies linear differential equation $\frac{\partial \varphi}{\partial t} = A(t)\varphi(t, \tau)$

2. $\varphi(t, t) = I$

3. For any three time instants $\varphi(t_3, t_1) = \varphi(t_3, t_2) \varphi(t_2, t_1)$

Properties of STM

4. $\varphi(\tau, t) = [\varphi(t, \tau)]^{-1}$

5. For time-invariant systems

$$\varphi(0) = I$$

$$\varphi(t)\varphi(\tau) = \varphi(t + \tau)$$

$$\varphi^{-1}(t) = \varphi(-t)$$

6. For linear time invariant system

$$\varphi(t, \tau) = e^{A(t-\tau)}$$

Solution of Linear Time Varying Systems

Solution: $X(t) = \varphi(t, t_0) C(t)$ (Method of variation of parameters)

How to determine $C(t)$?

$$\dot{X} = AX + BU$$

$$\left[\frac{\partial \varphi}{\partial t} C \right] + \varphi \dot{C} = [A\varphi C] + BU, \quad \dot{C} = [\varphi(t, t_0)]^{-1} BU$$

$$C(t) = C(t_0) + \int_{t_0}^t \left[\varphi^{-1}(\tau, t_0) \right] B(\tau) U(\tau) d\tau$$

$$X(t_0) = C(t_0), \quad \varphi(t_0, t_0) = I$$

Solution of Linear Time Varying Systems

$$\begin{aligned} X(t) &= \varphi(t, t_0) \left[X(t_0) + \int_{t_0}^t \varphi^{-1}(\tau, t_0) B(\tau) U(\tau) d\tau \right] \\ &= \varphi(t, t_0) X(t_0) + \int_{t_0}^t [\varphi(t, t_0) \varphi(t_0, \tau)] B(\tau) U(\tau) d\tau \\ X(t) &= \varphi(t, t_0) X(t_0) + \int_{t_0}^t \varphi(t, \tau) B(\tau) U(\tau) d\tau \end{aligned}$$

Thanks for the Attention...!

