

Lecture – 2
*First and Second Order
Linear Differential Equations*

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Linear Differential Equations

Principle of superposition:

If $x_1(t)$ and $x_2(t)$ are solutions then $\alpha_1 x_1(t) + \alpha_2 x_2(t)$ is also a solution for any scalars α_1 and α_2

Example:

$$\ddot{x} + x = 1 \quad (\text{Non homogeneous})$$

$x_1 = 1 + \cos t$ and $x_2 = 1 + \sin t$ both are solutions.

But

$2(1 + \cos t)$ or $[(1 + \cos t) + (1 + \sin t)]$ are not solutions.

Linear Differential Equations

However, if

$$\ddot{x} + \dot{x} = 0 \quad (\text{Homogeneous})$$

then both of the above are solutions.

Lesson: Principle of superposition holds good only for homogeneous linear differential equations.

Homogeneous First-order Equations

System dynamics: $\dot{x} + kx = 0$, $x(t_0) = x_0$, $k = 1/\tau$

Solution: $(dx/x) = -k dt$

$$\ln x = -k t + \ln c$$

$$\ln(x/c) = -k t$$

$$x = e^{-kt} c$$

Initial condition: $x_0 = e^{-kt_0} c$, $c = e^{kt_0} x_0$

Hence,

$$x(t) = e^{-k(t-t_0)} x_0$$

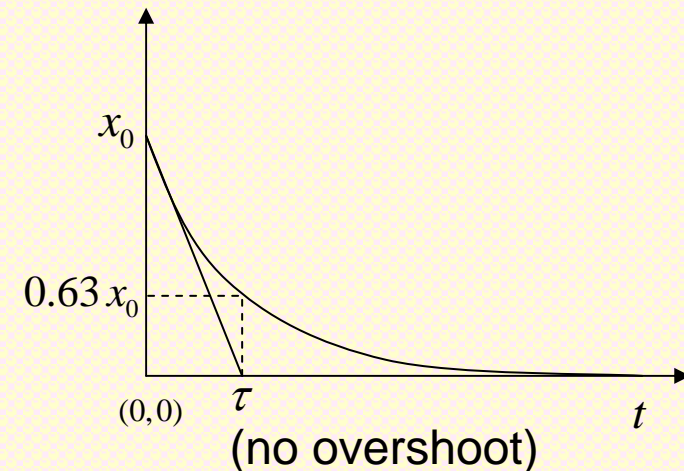
Homogeneous First-order Equations

Note:

$$1) \quad e^{at} = 1 + at + \frac{a^2 t^2}{2!} + \frac{a^3 t^3}{3!} + \dots$$

2) If $t_0 = 0$, then the solution is

$$x(t) = e^{-kt} x_0 = e^{-(t/\tau)} x_0$$



Non-homogeneous First-order Equations: Response to Step Inputs

Consider: $\dot{x} + \frac{1}{\tau}x = A$ (A: constant)

Homogeneous solution:

$$\dot{x}_h + \frac{1}{\tau}x_h = 0$$

$$x_h = e^{-\frac{t}{\tau}}c$$

Particular solution: $x_p = B$

Substitute: $\frac{1}{\tau}B = A$

$$\therefore x_p = B = \tau A$$

Non-homogeneous First-order Equations: Response to Step Inputs

$$x(t) = x_h + x_p$$

$$x(t) = e^{-\frac{t}{\tau}} c + \tau A$$

$$x_0 = c + \tau A \quad \Rightarrow c = x_0 - \tau A$$

$$x(t) = e^{-\frac{t}{\tau}} (x_0 - \tau A) + \tau A$$

$$x_{ss} = \lim_{t \rightarrow \infty} x(t) = \tau A \quad (\text{Note: } x_{ss} \neq A)$$

$$\text{Complete solution: } x(t) = e^{-\frac{t}{\tau}} (x_0 - \tau A) + \tau A$$

Steady state error (*wrt.* input A):

$$e_{ss} = A - [x(t)]_{t \rightarrow \infty} = A - \tau A = (1 - \tau) A$$

First-order Equations in Special Form: Unit Step Response

Consider: $\dot{x} + \left(\frac{1}{\tau}\right)x = \left(\frac{1}{\tau}\right)A$ (Special case in text books: $A = 1$)

Homogeneous:

$$\dot{x}_h + \frac{1}{\tau}x_h = 0$$

$$x_h = e^{-\frac{t}{\tau}}c$$

Particular solution: $x_p = B$

Substitute: $\left(\frac{1}{\tau}B = \frac{1}{\tau}A\right) \Rightarrow (B = A)$

$$\therefore x_p = B = A$$

First-order Equations in Special Form: Unit Step Response

$$x(t) = x_h + x_p$$

$$x(t) = e^{-\frac{t}{\tau}} c + A$$

$$x_0 = c + A \quad \Rightarrow \quad c = x_0 - A$$

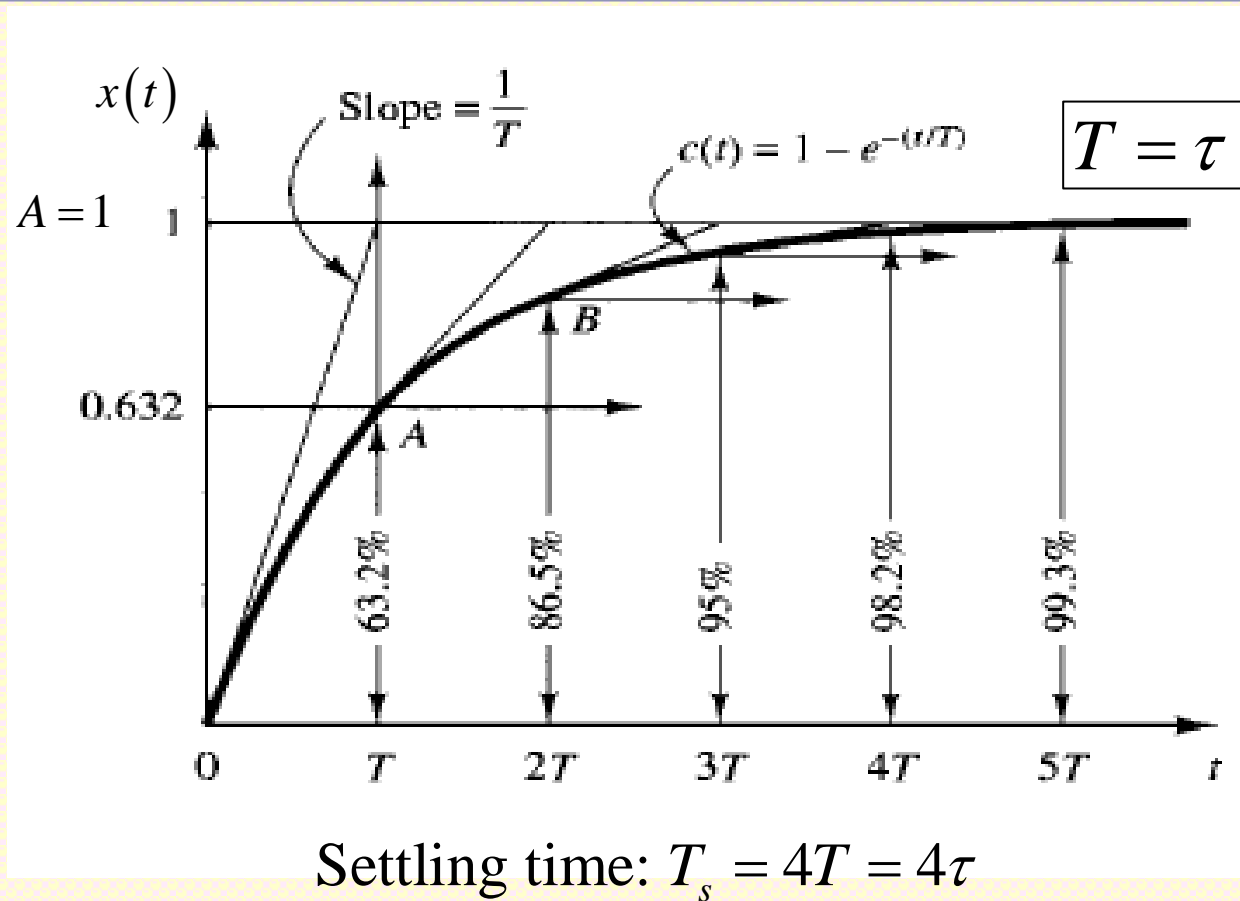
$$x(t) = e^{-\frac{t}{\tau}} (x_0 - A) + A$$

$$x_{ss} = \lim_{t \rightarrow \infty} x(t) = A$$

$$\text{Complete solution: } x(t) = e^{-\frac{t}{\tau}} (x_0 - A) + A$$

$$\text{Steady state error (wrt. input } A\text{): } e_{ss} = A - x_{ss} = A - A = 0$$

Unit Step Response of a First-Order System



Second Order System

Homogeneous:

$$\ddot{x} + a\dot{x} + bx = 0, \quad (a = 2\xi\omega_n, b = \omega_n^2)$$

Guess:

$$x(t) = e^{\lambda t}$$

$$(\lambda^2 + a\lambda + b)e^{\lambda t} = 0$$

$$e^{\lambda t} \neq 0$$

$$\Rightarrow \lambda^2 + a\lambda + b = 0 \quad : \text{characteristic equation}$$

$$\lambda_{1,2} = \frac{-a \pm \sqrt{a^2 - 4b}}{2}$$

Second Order System

Note: $a, b \in \mathbb{R}$

\Rightarrow Arises three cases

Case 1: Distinct and real roots

Case 2: Complex conjugate roots

Case 3: Real and repeated roots

Case-1: Distinct and real roots

$$x_1 = e^{\lambda_1 t}, \quad x_2 = e^{\lambda_2 t}$$

$$x(t) = c_1 x_1(t) + c_2 x_2(t)$$

$$x(t) = c_1 e^{\lambda_1 t} + c_2 e^{\lambda_2 t}$$

Case-2: Complex conjugate roots

$$\lambda_1 = \sigma + j\omega, \quad \lambda_2 = \sigma - j\omega$$

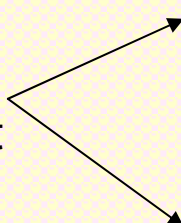
$$x_1 = e^{(\sigma + j\omega)t} = e^{\sigma t} (\cos \omega t + j \sin \omega t)$$

$$x_2 = e^{(\sigma - j\omega)t} = e^{\sigma t} (\cos \omega t - j \sin \omega t)$$

$$x(t) = c_1 e^{(\sigma + j\omega)t} + c_2 e^{(\sigma - j\omega)t}$$

However notice that:

Linearly independent terms


$$\frac{x_1 + x_2}{2} = e^{\sigma t} \cos \omega t$$
$$\frac{x_1 - x_2}{2j} = e^{\sigma t} \sin \omega t$$

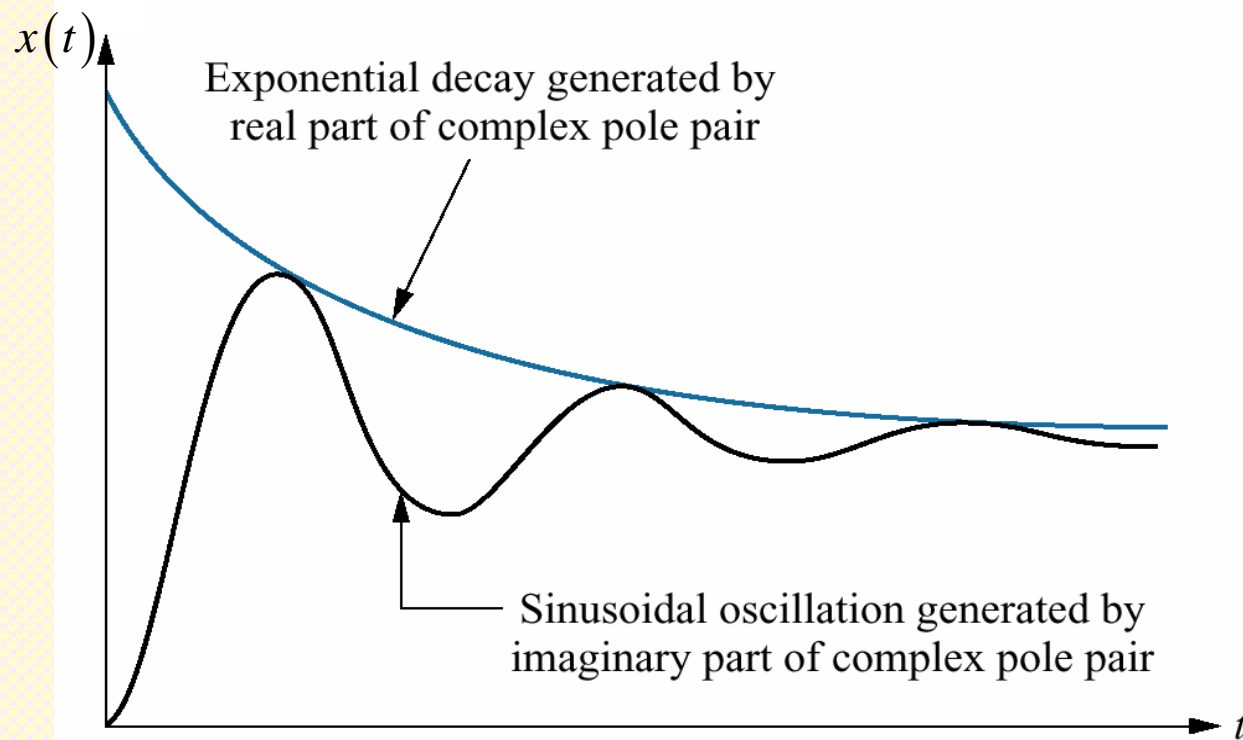
Case-2: Complex conjugate roots

So, we can also write the solution as :

$$x(t) = A \left(\frac{x_1 + x_2}{2} \right) + B \left(\frac{x_1 - x_2}{2j} \right)$$

$$x(t) = e^{\sigma t} (A \cos \omega t + B \sin \omega t)$$

Complex conjugate roots: Typical response plot



Case-3: Real and repeated roots

$$\lambda_1 = \lambda_2 = \lambda$$

$$x_1(t) = e^{\lambda t}$$

How to find $x_2(t)$

Method of variation of parameters

Assume: $x_2(t) = u(t) x_1(t)$

Then

$$\dot{x}_2 = u \dot{x}_1 + \dot{u} x_1$$

$$\ddot{x}_2 = u \ddot{x}_1 + \dot{u} \dot{x}_1 + \dot{u} \dot{x}_1 + \ddot{u} x_1$$

Case-3: Real and repeated roots

Substituting back and rearranging the terms, it leads to

$$u (\ddot{x}_1 + a \dot{x}_1 + b) + \dot{u} (2\dot{x}_1 + a x_1) + \ddot{u} x_1 = 0$$

However, $(\ddot{x}_1 + a \dot{x}_1 + b) = 0$ (since x_1 is a solution)

Moreover, $\dot{x}_1 = \lambda e^{\lambda t} = \lambda x_1 = -\left(\frac{a}{2}\right) x_1$ (since $\lambda = -\frac{a}{2}$)

$$(2\dot{x}_1 + a x_1) = 0$$

This leads to $\ddot{u} x_1 = 0$. Moreover $x_1 = e^{\lambda t} \neq 0$. Hence $\ddot{u} = 0$

Case-3: Real and repeated roots

One solution $u(t) = t$

$$\therefore x_2(t) = t x_1(t) = t e^{\lambda t}$$

$$x(t) = c_1 x_1(t) + c_2 x_2(t)$$

$$x(t) = (c_1 + c_2 t) e^{\lambda t}$$

Corollary: When $\lambda = 0$ (double poles at origin)

$$x(t) = c_1 + c_2 t \rightarrow \infty \quad \text{as } t \rightarrow \infty$$

Double pole (in general, multiple poles) at origin is de-stabilizing..!

Summary

Case	Roots of characteristic equation	Basics	General Solution
1	Distinct & Real λ_1, λ_2	$e^{\lambda_1 t}, e^{\lambda_2 t}$	$c_1 e^{\lambda_1 t} + c_2 e^{\lambda_2 t}$
2	Complex Conjugate $\lambda = \sigma + j\omega$	$e^{\sigma t} \cos \omega t$ $e^{\sigma t} \sin \omega t$	$e^{\sigma t} (A \cos \omega t + B \sin \omega t)$
3	Real double Roots	$e^{\lambda t}, t e^{\lambda t}$	$(c_1 + c_2 t) e^{\lambda t}$

Example

$$\ddot{x} - 4\dot{x} + 4x = 0 \quad x(0) = 3, \quad \dot{x}(0) = 1$$

$$\lambda^2 - 4\lambda + 4 = 0 \quad \lambda = 2, 2 \quad (\text{Real, repeated})$$

$$x(t) = (c_1 + c_2 t) e^{2t}$$

$$\dot{x}(t) = 2(c_1 + c_2 t) e^{2t} + c_2 e^{2t}$$

Solving for the initial conditions $c_1 = 3$ $c_2 = -5$

$$x(t) = (3 - 5t) e^{2t}$$

Particular Solution: Method of Undetermined Coefficients

$$\ddot{x} + ax + bx = f(t)$$

Terms in $f(t)$	Choice of $x_p(t)$
$k e^{pt}$	$c e^{pt}$
$k t^n$	$k_n t^n + k_{n-1} t^{n-1} + \dots + k_1 t + k_0$
$k \cos \omega t$	$A \cos \omega t + B \sin \omega t$
$k \sin \omega t$	$A \cos \omega t + B \sin \omega t$

Substitute these expressions back in the differential equation and determine the unknown coefficients

Example

Find $x_p(t)$ for $\ddot{x} - 3\dot{x} + 2x = 4t + e^{3t}$

Choose $x_p(t) = (k_1t + k_0) + ce^{3t}$

Substitute and equate the coefficients

$$k_1 = 2, \quad k_0 = 3, \quad c = \frac{1}{2}$$

$$\therefore x_p(t) = (2t + 3) + \frac{1}{2}e^{3t}$$

Second order system in Standard form

$$\ddot{c} + 2\zeta\omega_n\dot{c} + \omega_n^2c = \omega_n^2u$$

ω_n : Natural frequency

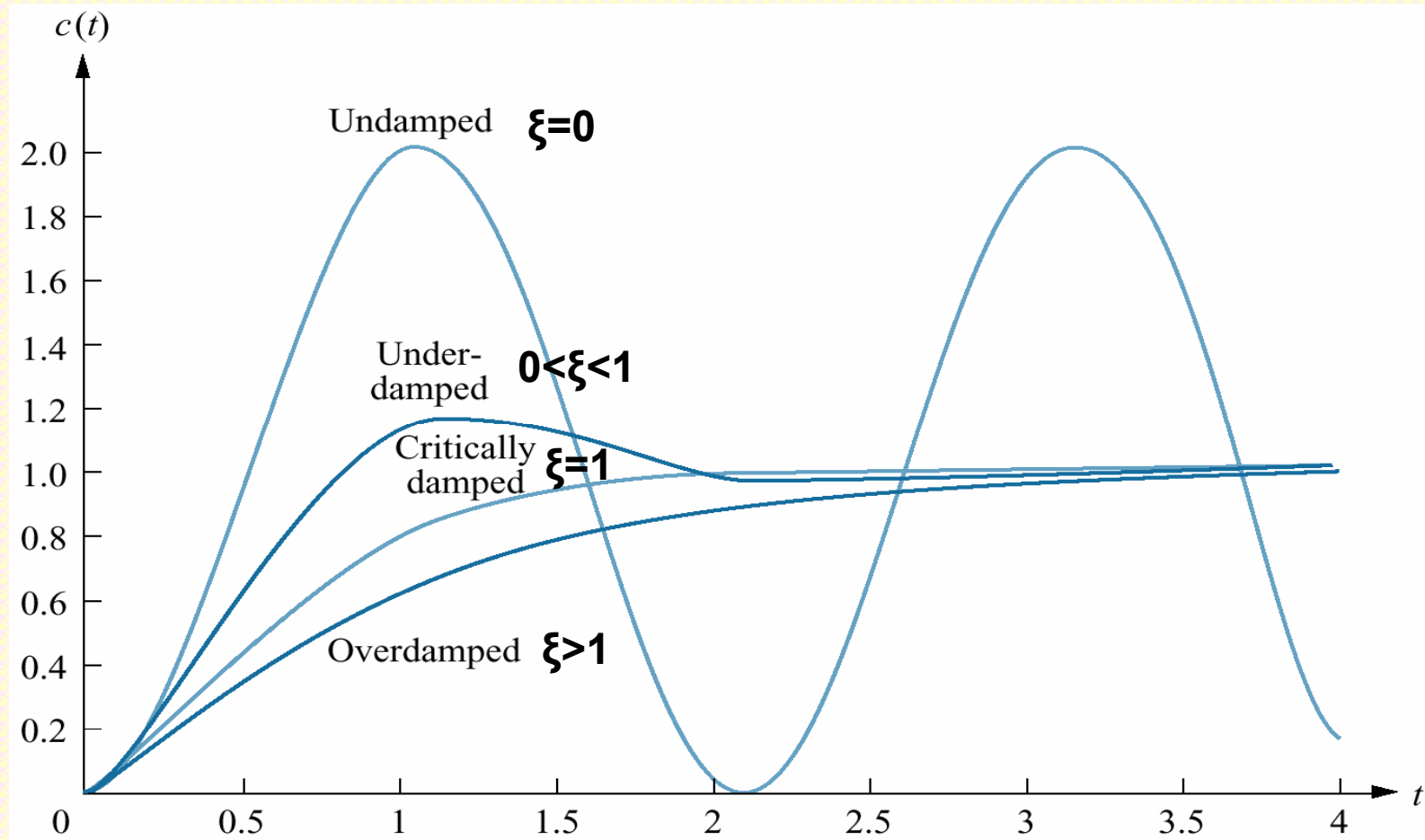
ζ : Damping ratio

Transfer function (will be studied later):

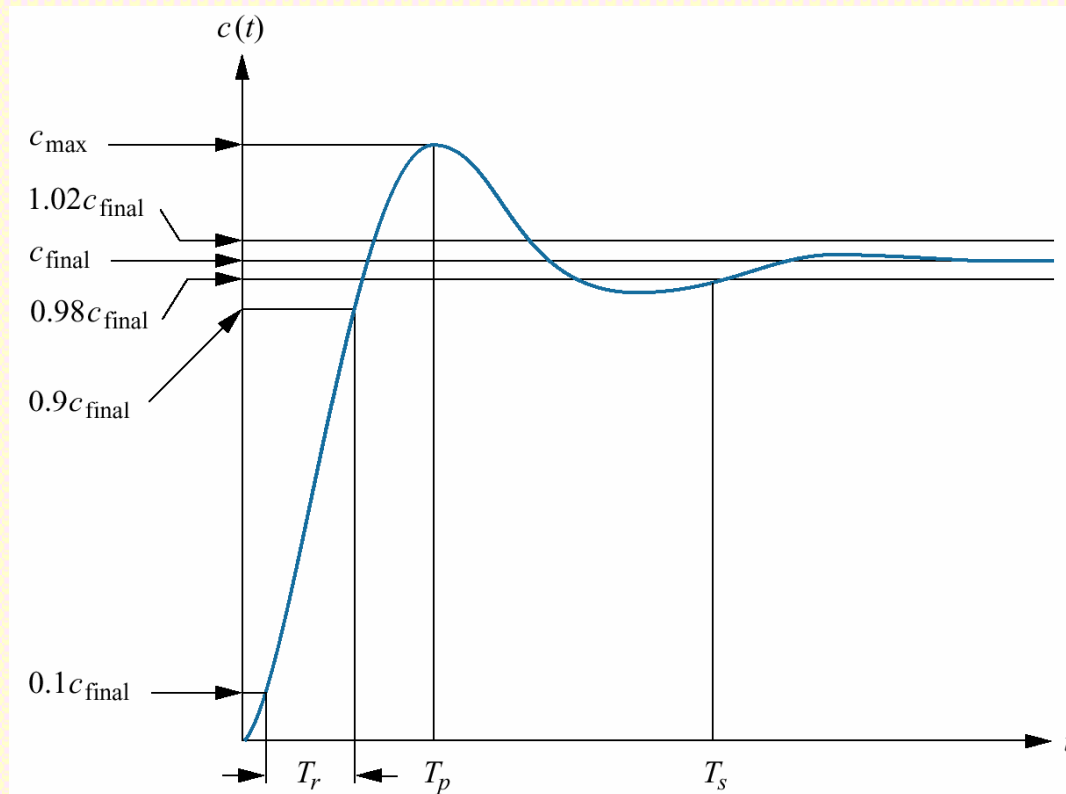
$$\frac{C(s)}{U(s)} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

Roots: poles

Unit Step Response of a Second-order System



Under-damped system response specifications



Ref: N. S. Nise:
Control Systems Engineering,
4th Ed., Wiley, 2004

T_r : Rise time

T_p : Pick time

T_s : Settling time

Transient Response Specifications

$$\text{Rise time: } T_r = \frac{\pi - \beta}{\omega_d}, \quad \text{Peak time: } T_p = \frac{\pi}{\omega_d}$$

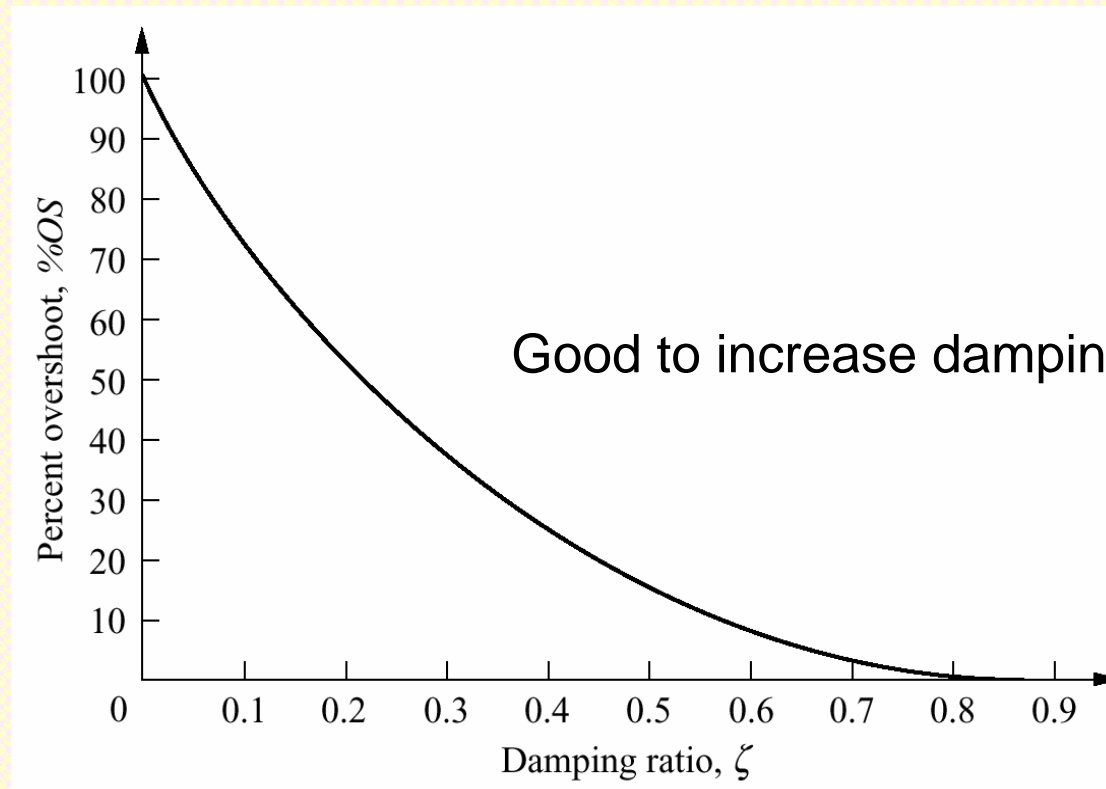
$$\text{Damped natural frequency: } \omega_d = \omega_n \sqrt{1 - \zeta^2}$$

$$\text{Maximum overshoot: } M_p = e^{\left(\frac{-\pi\zeta}{\sqrt{1-\zeta^2}} \right)}$$

$$\text{Settling time: } T_s = \frac{4}{\zeta\omega_n} \quad (2\% \text{ criterion})$$

$$= \frac{3}{\zeta\omega_n} \quad (5\% \text{ criterion})$$

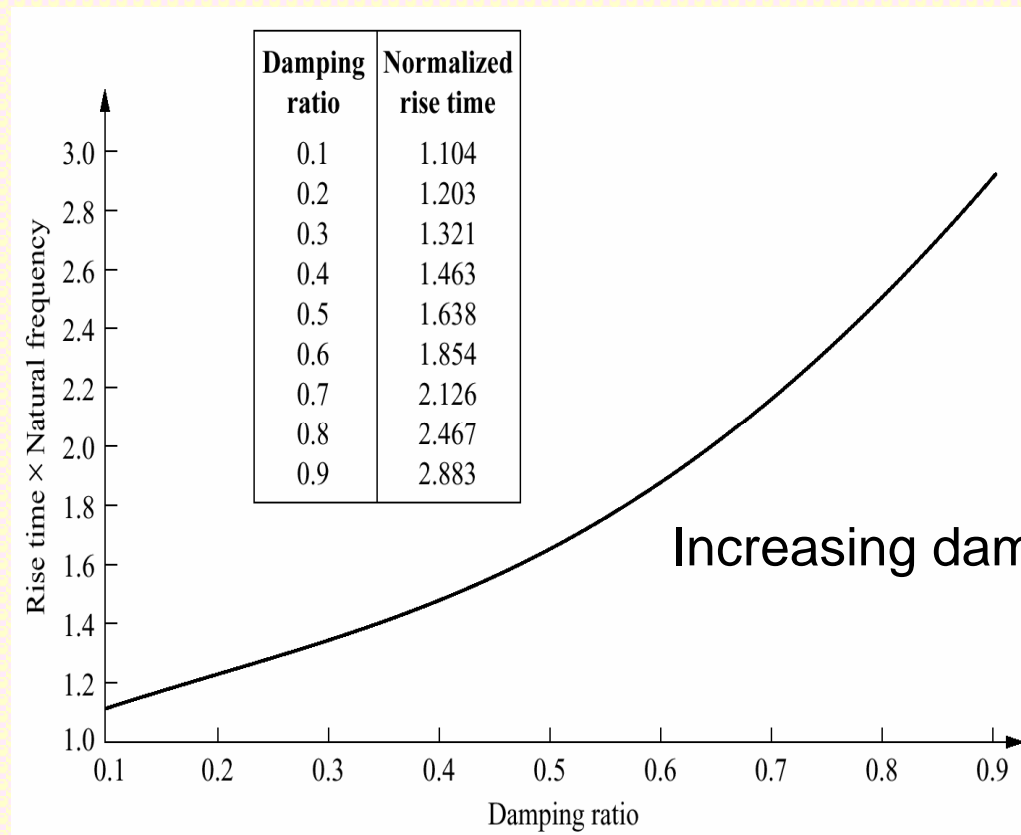
Percentage over-shoot as a function of damping ratio



Ref: N. S. Nise:
Control Systems Engineering,
4th Ed., Wiley, 2004

Good to increase damping ratio.

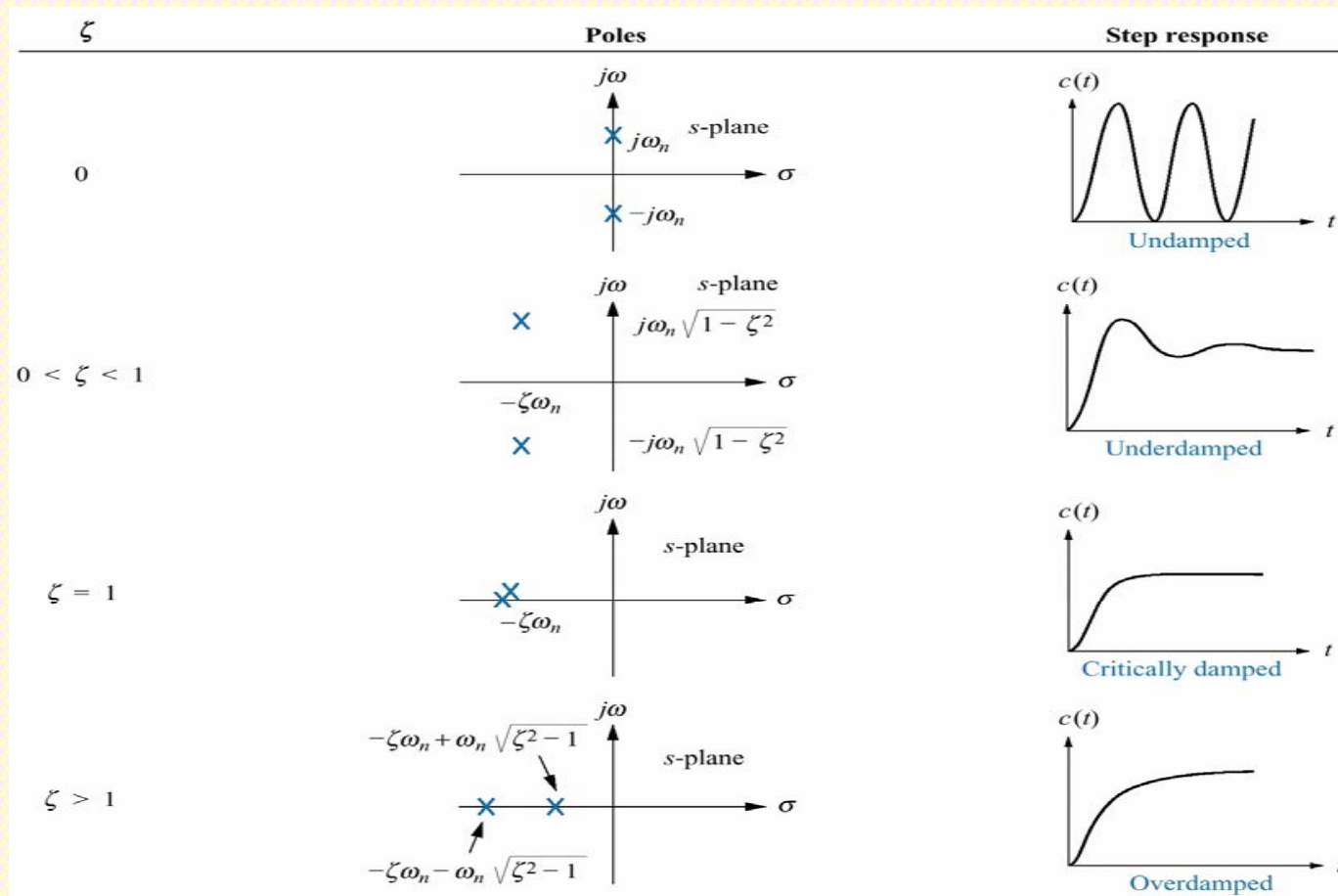
Rise time as a function of damping ratio



Ref: N. S. Nise:
Control Systems Engineering,
4th Ed., Wiley, 2004

Increasing damping ratio too much is Bad..!

Second-order System Response as a Function of Damping Ratio



Thanks for the Attention...!



